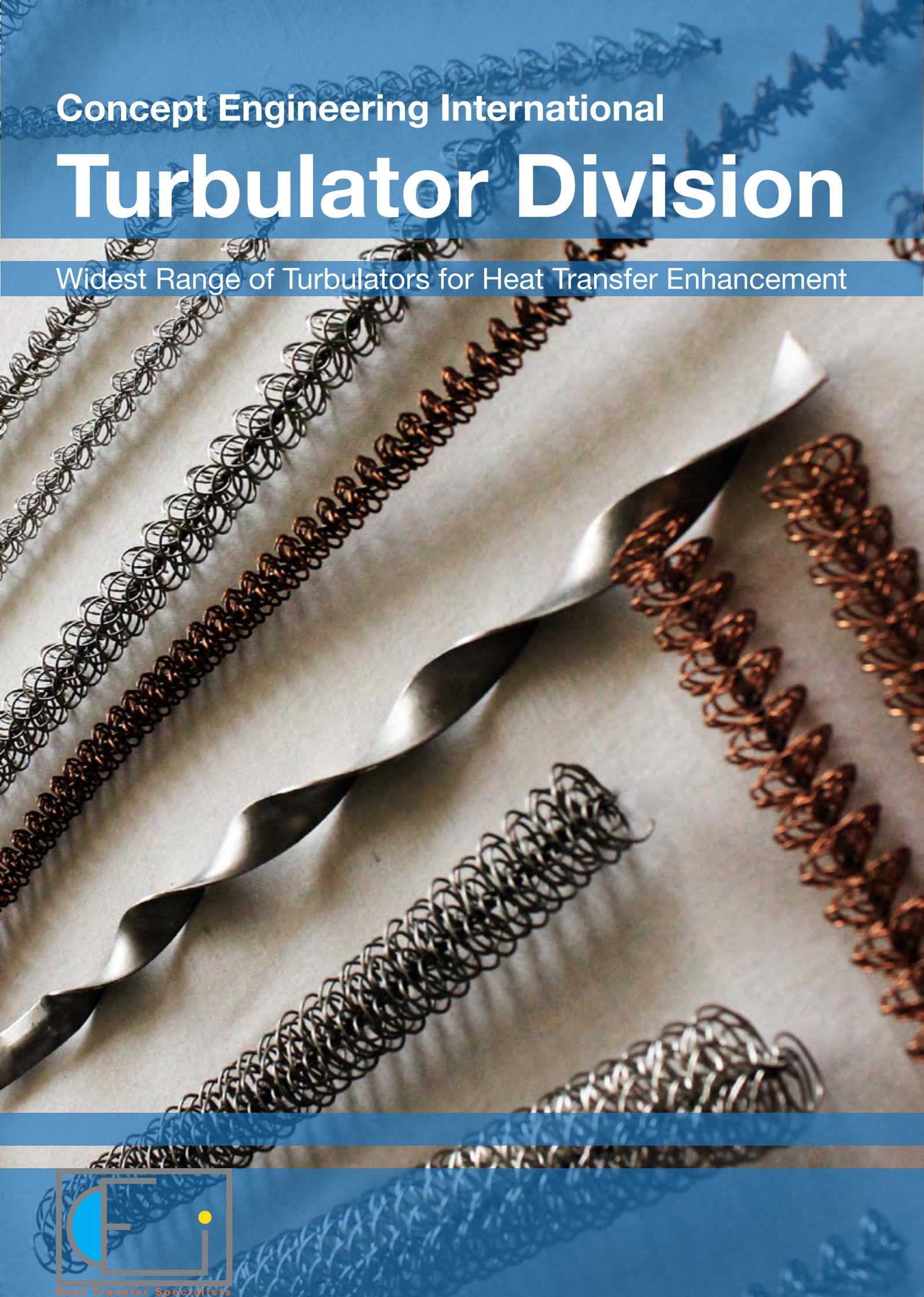


Concept Engineering International

Turbulator Division

Widest Range of Turbulators for Heat Transfer Enhancement



Heat Transfer Specialists

The Turbulator Division

Concept Engineering International is a heat transfer focused engineering company with over 90% of its products exported to developed markets including Austria, USA, Australia, UK, Germany, Singapore, Malaysia, Indonesia, Thailand, UAE, Mexico among others. Our range of turbulators is indeed very wide and covers most turbulator types.

Possibilities offered by Turbulators:

1. Correctly used turbulators can make equipment a tenth of its normal size in some instances. In others it makes it significantly smaller. Basically its use is warranted in the following instances:

- a. Where the fluid in the tube is viscous.
- b. The fluid flow in the tube has a low Reynolds number.
- c. All kinds of oil coolers and thermic heaters.
- d. Gasses in the tube.

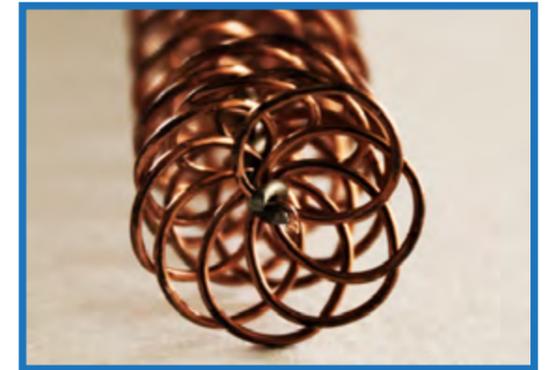


Product and design philosophy

1. We have developed our core products using wire due to its low pressure drop high turbulence characteristics making them the best class of turbulators in the world.
2. Since customers may require other types of turbulators due to legacy issues, user specifications or specialized application, we make other types also. Together we have a large range and pride ourselves on being a one stop shop for turbulators.

Major Types of turbulators

1. Flexible or wire Petal turbulators. (Also called wire matrix turbulators)
2. Rigid center rod wire turbulators. (sometimes soldered to tubes)
3. Finned Hollow Rod type turbulators.
4. Twisted tape turbulators. (Traditional).



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Major Applications for Turbulators:

1. Oil Coolers
2. Highly viscous liquids
3. Gas or Air heaters/coolers
4. Static Mixers
5. Falling Film Evaporators
6. Inline reactors
7. Prevention of scale formation on tube walls and reduction of fouling by creation of turbulence.

Basic Principles behind turbulators:

For Heat exchangers

1. A fluid flowing through a tube unless in turbulent flow tends to form a film at the tube wall which impedes heat transfer.
2. A Turbulator breaks this film and makes the flow turbulent raising the effective Reynolds number. This increases heat transfer in the case of heat exchangers, as well as induces mixing and reactions in inline mixers and reactors.
3. Wire due to its small diameter and cylindrical nature offers comparatively lower resistance while offering a large matrix for interacting with the fluid. It thus gives higher mixing and heat transfer for a lower pressure drop.
4. The turbulence created also greatly reduces fouling as particulate matter does not get deposited on the tube walls but is swept off by the turbulence created.

Advantages of Flexible or wire petal turbulators.

(Also called wire matrix turbulators)

- Light weight, flexible and cheap.
- Easy to install, remove and reinsert after cleaning.
- Very efficient.

Range

Sizes

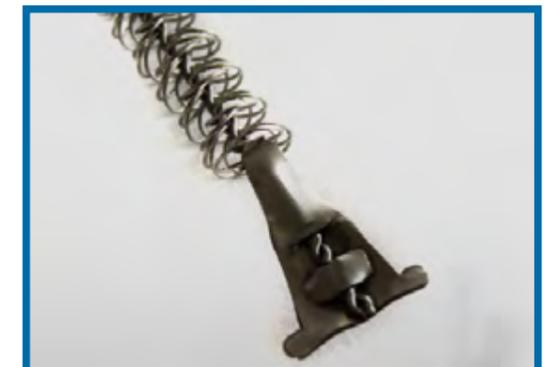
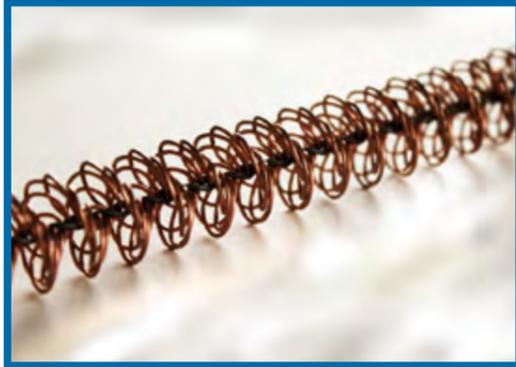
We can offer for all sizes of tubes from 1/4" to 1.25" OD.

Loop Density

For each tube OD we can offer a wide variety of Loop densities from Low density to Ultra High Density. Apart from our standard range customization of the density can also be done as per customer requirement.

Material of Construction

Stainless Steel 304 / 304L, 316 / 316L, 321 etc.
Carbon Steel.
Copper
Brass
Monel
Fecralloy



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Rigid center rod wire turbulators sometimes soldered to tubes.

It is possible to give a high wire loop density on a rigid central rod where the loops are soldered both to the central wire rod as well as if desired the tube wall. This arrangement has the following advantages/ disadvantages.

Advantages

- A very high concentration of loops can be put on the center rod as the loops are Oval rather than circular.
- If the loops are soldered inside the tube, they give very good bonding and so increase heat transfer through conduction in addition to through turbulence.
- If soldered inside tubes, this turbulator can withstand high pressure.
- Can be offered in most materials of construction except aluminium.

Disadvantages

- Higher cost.
- Once installed, cannot be removed. As turbulators brazed/soldered in tubes.
- Solder will melt at temperatures around 180 /290 degrees C for normal/High temperature solder. Hence not suitable for higher applications above 290 degrees C.

Finned Hollow Rod type turbulators.

For large diameter tubes, it is sometimes desirable to block the center of the tube, to narrow the passage. In such a case we can offer a hollow rod type Turbulator. In this case we substitute the rigid center rod for a small diameter tube the ends of which have been sealed. This sealing of the ends converts the small diameter tube into a light weight hollow rod. The loops are then soldered on just like in the Rigid center rod type Turbulator. This type of Turbulator is generally used in tubes where the id of the tube is more than one inch.

Twisted tape turbulators

The twisted tape is the old war horse of the Turbulator world and of course we make them in large quantities. This type is also featured in the HTRI software as a generic product so customers can do their own design. (A type of wire Turbulator is also featured but as a proprietary product of Calgavin and customized as per their configurations.)

We can give all standard and a large range of custom pitches and offer them in almost all materials.

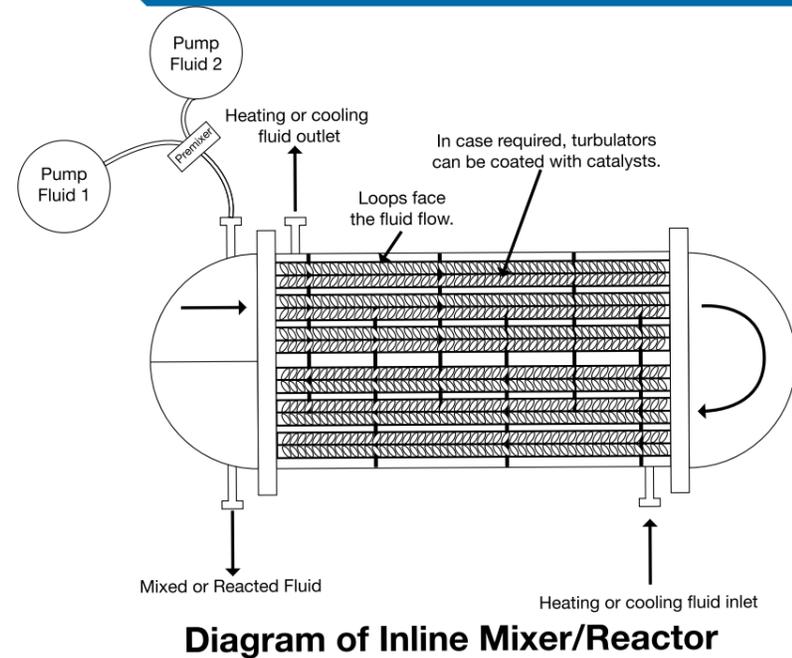
While in most cases the flexible wire type is a preferred option, in the case of retrofitting, where there is a lower flexibility with regards to redesigning the existing equipment, this is very often a low pressure drop reasonable efficiency solution.



Twisted Tape Turbulator



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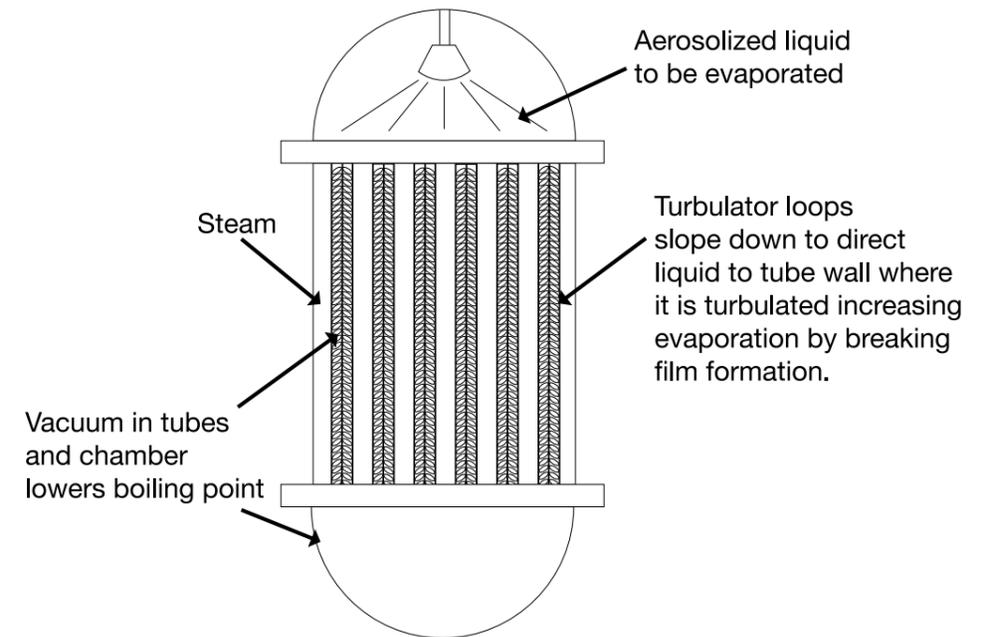


Turbulators in Static Mixers and Inline reactors.

1. Agitated vessels for mixing or reactions can be replaced by passing the fluids to be mixed through a set of turbulated tubes.

By this intube micromixing it is possible to achieve:

1. Online versus batch mixing and reactions.
2. Significantly lower equipment size.
3. Significantly Lower power consumption.
4. Significantly lower mixing time.
5. For reactions it is possible to coat the turbulators with catalysts or to make them from catalyzing metals.
6. For reactions it is possible to add or remove heat by housing the turbulated tubes within an exchanger where the heat transfer fluid (either heating or cooling) can be passed through the shell. (steam, hot/ cold water, Thermic oil.)
7. Turbulators are also very useful when fluids need to be mixed in the feed pipe itself before entering a process chamber.

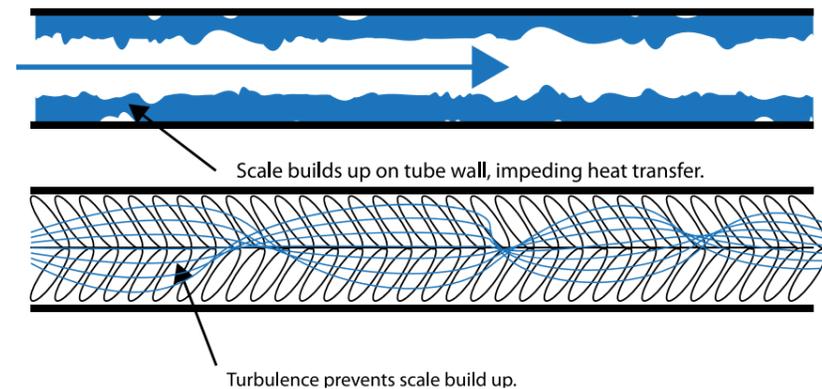


Turbulators in Falling Film evaporators

When placed correctly in a the tubes of a falling film evaporator the loops direct the falling liquid to the wall of the tube and also turbulates the film of liquid increasing heat transfer and evaporation.

The turbulators also provide additional surface area for the evaporation to take place.

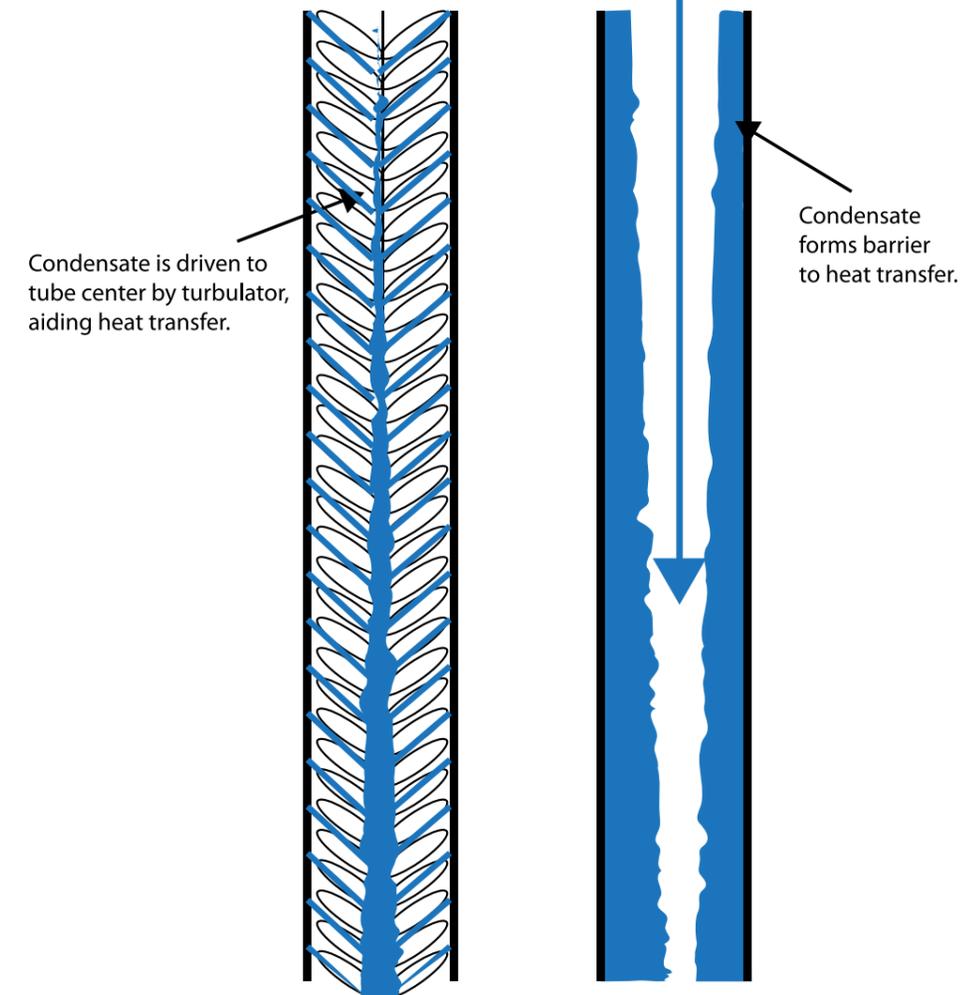
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Turbulators as scale inhibitors and removers

Scale formation occurs through deposition on the tube wall. This becomes more pronounced if the flow is laminar. This scale can choke off the tube leading to a huge efficiency drop.

1. Turbulators by disturbing the film along the wall of the tube, makes the flow turbulent and the scale is swept away before it can be settle. This significantly reduces scale, extending the time needed between tube cleanings.
2. Since the turbulators are removable with a strong twisted center wire, what scale is deposited can be easily dislodged by removing the turbulators. (By pulling them out with the help of the hook provided.)



Turbulators in vertical tube condensers and vertical tube steam air heaters

Where condensation happens inside vertical tubes, significant performance enhancement is achieved using flexible wire petal turbulators inserted with the loops facing upwards. This is because:

The condensate film forms a laminar layer impeding heat transfer. The Turbulator breaks the film and increases heat transfer and hence condensation. The condensate is diverted to the center of the tube removing its barrier forming potential significantly increasing the heat transfer at the tube wall. The contact points of the Turbulator and tube encourages drop formation at those points. These drops of condensate are drained away towards the center of the tube by the wire loops. This system is superior to putting the tubes horizontally as in such an arrangement the condensate settles at the floor of the tube reducing effective heat transfer area and in the case of steam condensate can cause knocking.

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Key elements of Good Turbulator design.

While we stand ready, and prefer to help our customers with design, we have tried to present an overview by presenting some graphical data.

To do this we have done the following:

Chosen a 3/4" OD tube as it is very widely used.

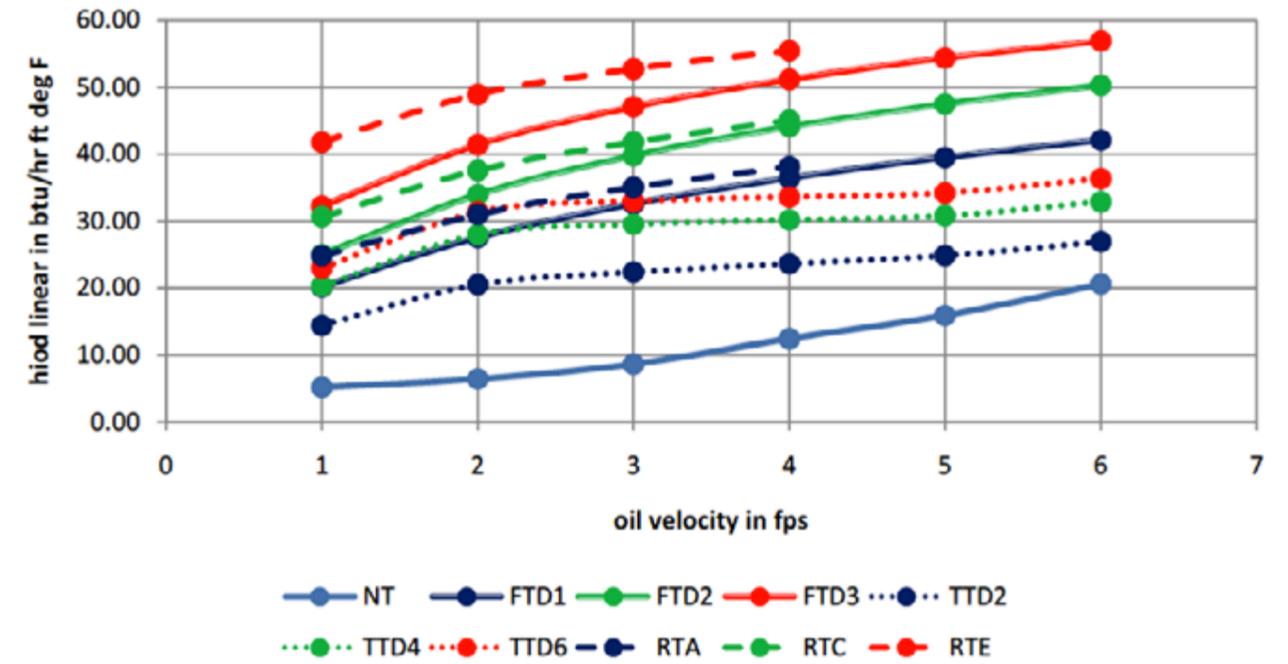
For this tube OD, we prepared the following data:

The pressure drop and heat transfer of oil of 3 viscosities, 5 CST, 15 CST and 25 CST travelling through the tubes at a flow rate of 1 to 6 feet per second. This data was prepared for the following

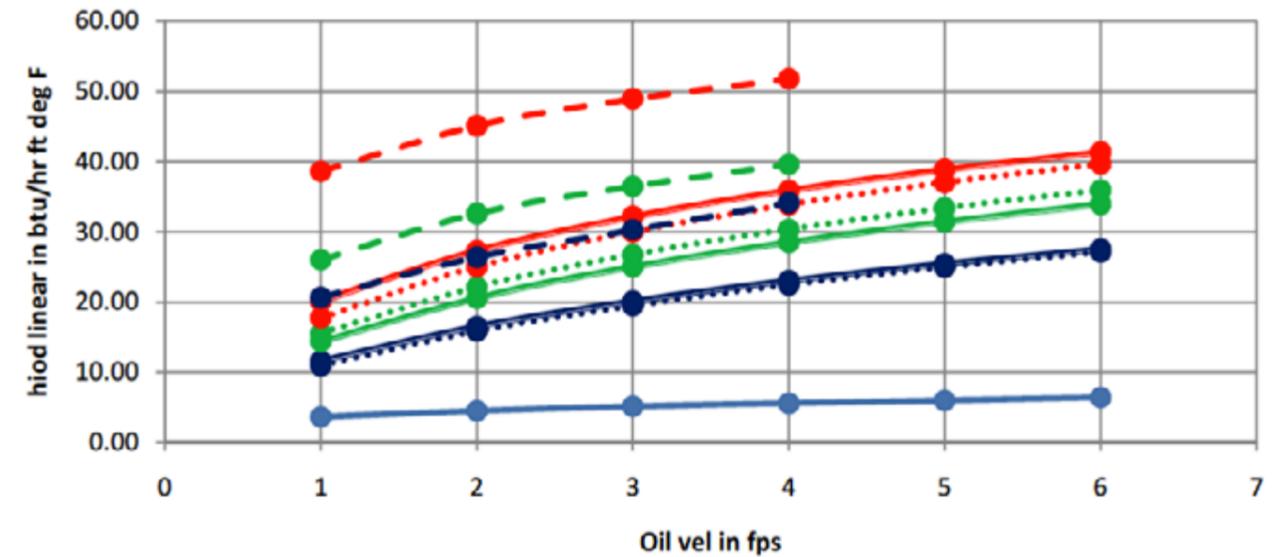
Turbulator cases:

- No Turbulator (NT)
- Rigid turbulators RTA, RTB & RTC. (Our standards for this tube size).
- Flexible turbulators FTD1, FTD2 & FTD3 (Again our standard models)
- Twisted tape turbulators. Since the possibilities here are infinite for the case of better comparing the two, we selected those twisted tape turbulators that gave with a 15 CST oil the same pressure drop at 3 FPS flow rate as the corresponding standard flexible turbulators. We must bear in mind though, that twisted tape turbulators thus selected have a very high twist ratio and are not commonly manufactured. However they were chosen for their value in comparison. Most commonly used twisted tape turbulators will have a lower performance and pressure drop.
- This data is represented graphically in the following charts/graphs.

hioid linear in btu/hr for No Turbulator, Rigid turbulator, Flexible turbulator & Twisted tape for 5 cSt oil viscosity

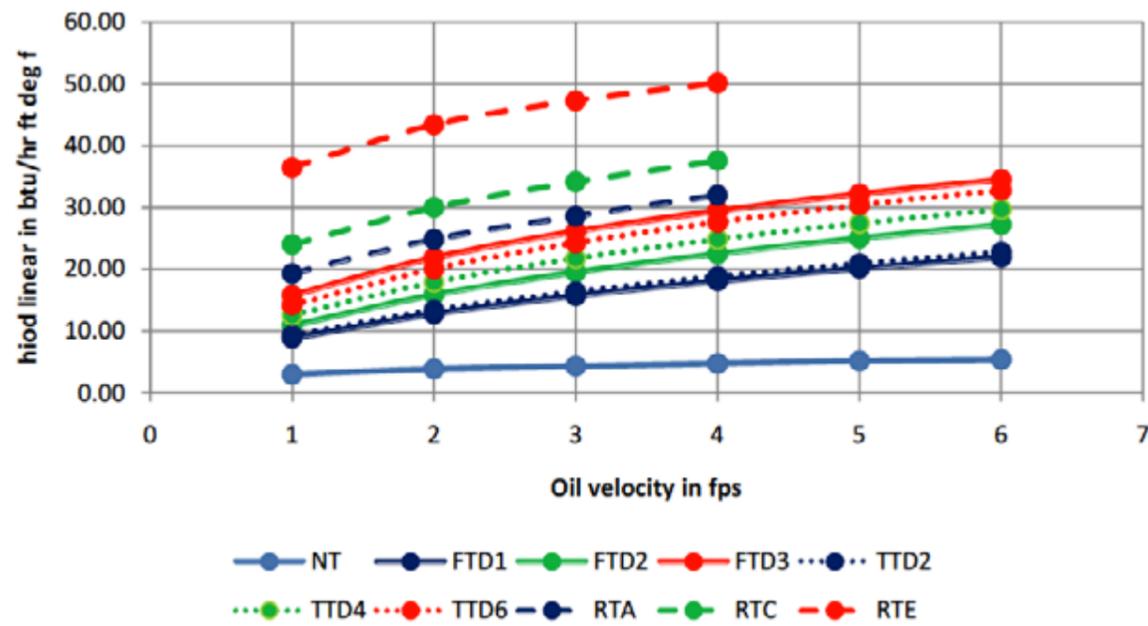


hioid linear in btu/hr for No Turbulator, Rigid turbulator, Flexible turbulator & Twisted tape for 15 cSt oil viscosity

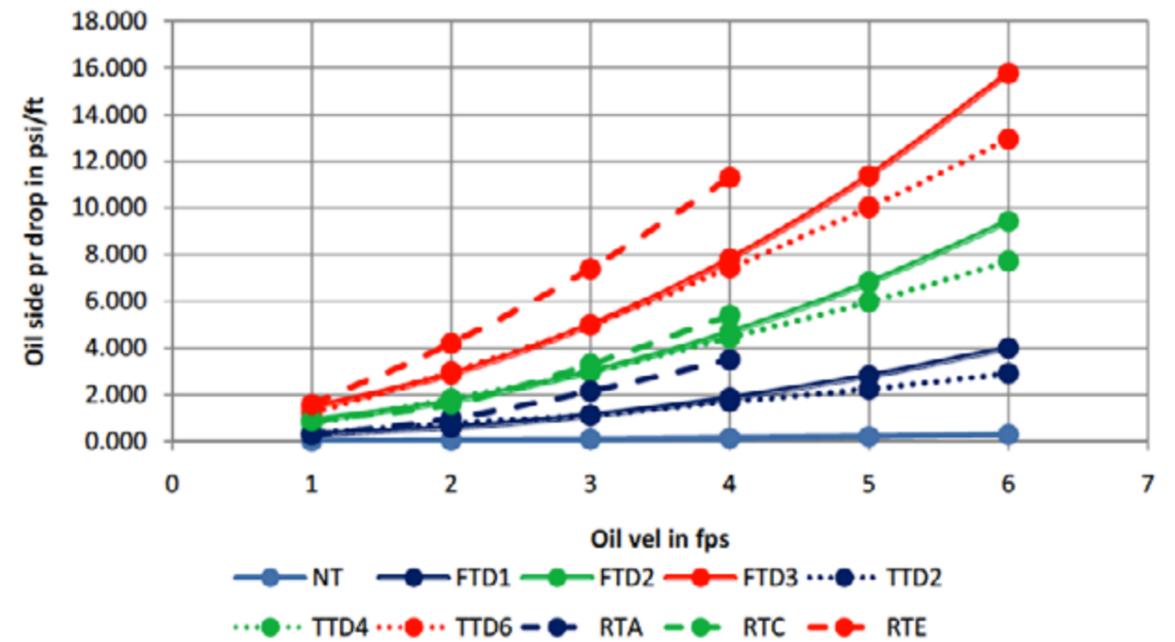


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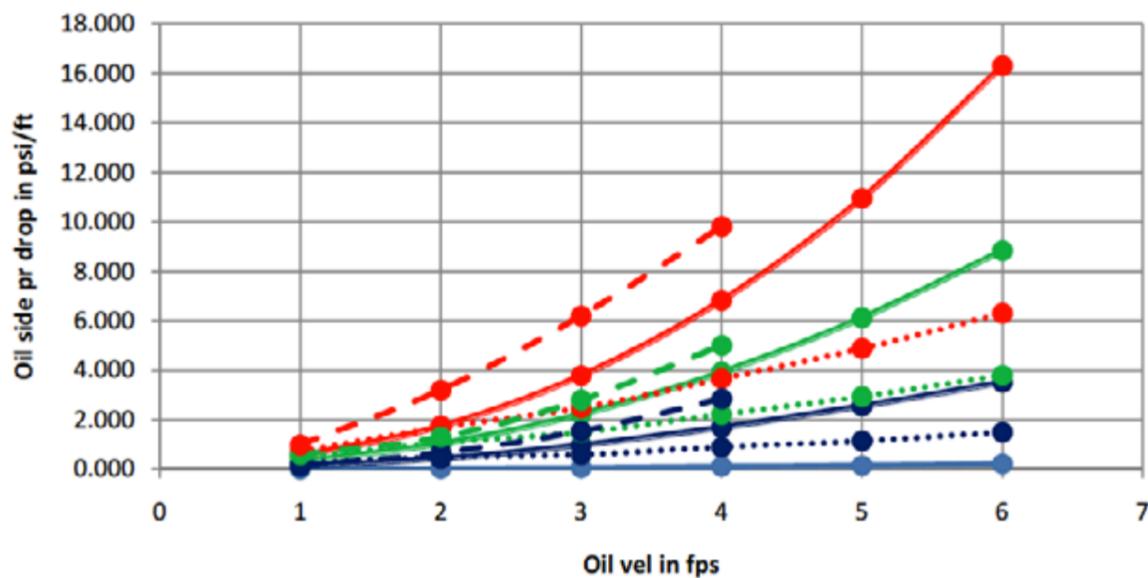
hiod linear in btu/hr for No Turbulator, Rigid turbulator, Flexible turbulator & Twisted tape for 25 cSt oil viscosity



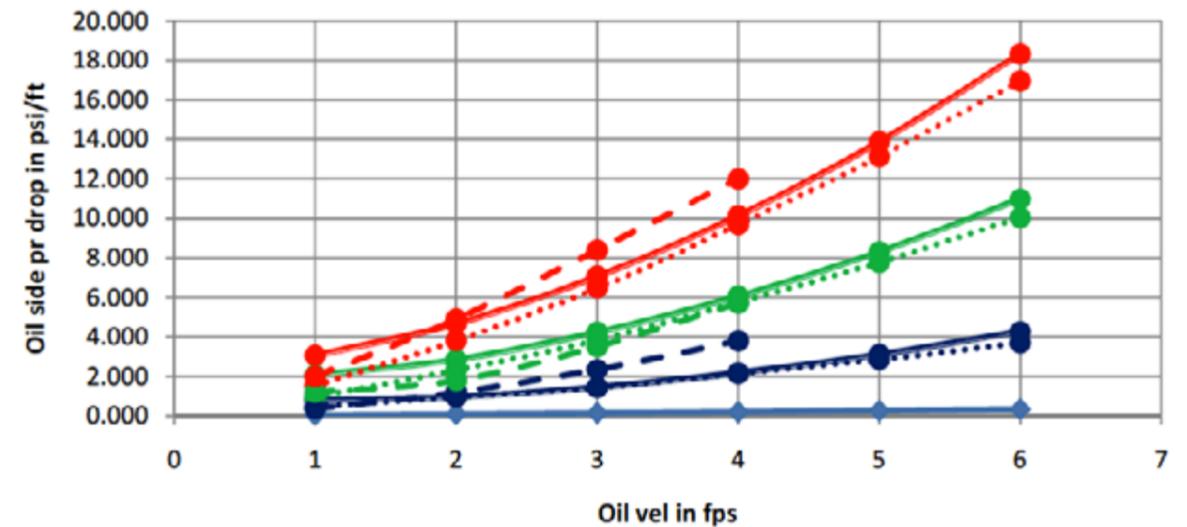
Oil side pr drop in psi/ft for No Turbulator, Rigid turbulator, Flexible turbulator & Twisted tape for 15 cSt oil viscosity



Oil side pr drop in psi/ft for No Turbulator, Rigid turbulator, Flexible turbulator & Twisted tape for 5 cSt oil viscosity



Oil side pr drop in psi/ft for No Turbulator, Rigid turbulator, Flexible turbulator & Twisted tape for 25 cSt oil viscosity



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From the curves, we can arrive at the following observations:

1) The rigid soldered turbulators overall give the best heat transfer and also the highest pressure drop. However as the viscosity goes up, the performance as compared to flexible turbulator goes up significantly. For example in the 5 cSt case, RT6 is above FTD3, RT4 above FTD2 and RT2 above FTD1. However at 15 cSt both RT6 and RT4 are above FTD 3 and RT2 is very close to FTD 3. In the case of 25 cSt all the three rigid turbulators are all above the highest performing flexible turbulators. The pressure drop is also compared to performance much lower.

2) The performance of the twisted tape (even though the twisting selected is high to match the pressure drop of corresponding flexible wire Turbulator) is generally lower than that of the flexible but not by much. However given that we will not have such a high degree of twisting in standard available twisted tape turbulators we can say that the flexible turbulators perform better.

3) The pressure drop increase for the two wire type turbulators with increase in fluid velocity is more than linear. The increase in performance is less than linear. This tells us that after a point it is not worth trying to purchase performance with pressure drop.

4) The performance as well as the pressure drop in the case of twisted tape turbulators are more linear.

To get another angle on this performance we worked out the following table, where we have simply divided the heat transfer coefficient by the corresponding pressure drop to get the HTPD factor (Heat transfer coefficient per unit of pressure drop). We did this for the three oil viscosities for all the Turbulator models at all the studied flowrates. We have color coded the results so that it is easier to spot the best as well as the trend.

| HTPD factors for Flexible, Twisted tape & Rigid turbulators | | | | | | | | | | | | |
|--|-------|-------|------|------|--|--------|------|------|--|-------|--------|------|
| vel fps | NT | FTD1 | FTD2 | FTD3 | | TTD2 | TTD4 | TTD6 | | RTA | RTC | RTE |
| 1 | 500.5 | 161.5 | 75.2 | 58.0 | | 55.2 | 41.3 | 31.2 | | 155.3 | 51.1 | 41.8 |
| 2 | 195.5 | 62.0 | 32.4 | 23.6 | | 40.2 | 25.9 | 18.4 | | 45.6 | 28.9 | 15.3 |
| 3 | 132.3 | 33.2 | 17.8 | 12.4 | | 37.2 | 19.7 | 13.3 | | 22.6 | 14.9 | 8.5 |
| 4 | 112.9 | 21.4 | 11.2 | 7.5 | | 26.1 | 13.6 | 9.2 | | 13.4 | 9.0 | 5.7 |
| 5 | 100.8 | 15.4 | 7.8 | 5.0 | | 21.5 | 10.5 | 7.0 | | | | |
| 6 | 93.6 | 12.0 | 5.7 | 3.5 | | 17.9 | 8.6 | 5.8 | | | 5 cSt | |
| | | | | | | | | | | | | |
| vel fps | NT | FTD1 | FTD2 | FTD3 | | TTD2 | TTD4 | TTD6 | | RTA | RTC | RTE |
| 1 | 119.4 | 37.5 | 15.9 | 13.9 | | 29.6 | 20.2 | 14.7 | | 72.5 | 28.9 | 24.1 |
| 2 | 72.2 | 27.3 | 11.9 | 9.5 | | 20.5 | 12.1 | 8.4 | | 26.9 | 20.4 | 10.7 |
| 3 | 53.6 | 17.9 | 8.4 | 6.4 | | 17.3 | 9.0 | 6.0 | | 14.1 | 11.0 | 6.6 |
| 4 | 38.1 | 12.4 | 6.1 | 4.6 | | 13.2 | 6.8 | 4.6 | | 9.8 | 7.3 | 4.6 |
| 5 | 27.9 | 9.0 | 4.6 | 3.4 | | 11.1 | 5.6 | 3.7 | | | | |
| 6 | 21.7 | 6.9 | 3.6 | 2.6 | | 9.3 | 4.7 | 3.1 | | | 15 cSt | |
| | | | | | | | | | | | | |
| vel fps | NT | FTD1 | FTD2 | FTD3 | | TTD2 | TTD4 | TTD6 | | RTA | RTC | RTE |
| 1 | 57.9 | 11.1 | 5.2 | 5.2 | | 20.1 | 13.0 | 9.2 | | 52.2 | 20.0 | 18.2 |
| 2 | 39.0 | 13.2 | 5.5 | 4.7 | | 13.9 | 7.7 | 5.3 | | 20.7 | 16.7 | 8.9 |
| 3 | 27.8 | 10.7 | 4.6 | 3.7 | | 11.4 | 5.7 | 3.8 | | 12.2 | 9.8 | 5.6 |
| 4 | 23.5 | 8.3 | 3.7 | 2.9 | | 8.7 | 4.3 | 2.9 | | 8.4 | 6.6 | 4.2 |
| 5 | 19.3 | 6.5 | 3.0 | 2.3 | | 7.3 | 3.5 | 2.3 | | | | |
| 6 | 15.6 | 5.1 | 2.5 | 1.9 | | 6.1 | 3.0 | 1.9 | | | 25 cSt | |
| | | | | | | | | | | | | |
| Note : HTPD factor is calculated for light , medium and dense turbulators as heat transfer coefficient hiod linear per unit pr drop at different oil viscosities and different oil velocities through tubes | | | | | | | | | | | | |
| | | | | Good | | Better | | Best | | | | |

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The following facts emerge:

- 1) Across all Turbulator types the HTPD factor goes down as the velocity increases.
- 2) Across all turbulator types the HTPD factor goes down as the winding density increases.
- 3) Different Turbulator types give comparatively different results under different viscosity and flow conditions.
- 4) We can conclude that the best Turbulator is case specific.

The following should be kept in mind:

- 1) The data for rigid turbulators are for rigid soldered turbulators. If the same turbulators are not soldered they perform but not as well. This is due to the better contact and so better heat transfer due to soldering.
- 2) Just as pressure drop is a cost we pay for performance so is the dollar cost of material and labour. The rigid soldered Turbulator while an extremely efficient Turbulator is expensive to make and install.
- 3) Ease of cleaning and maintenance are also key factors in turbulator choice.
- 4) Since we make all types, we have nothing to lose in giving the client correct advice.

Design

We can assist with the design data for the turbulators we produce on a case by case basis depending on the application, type of turbulators requested and the fluids involved.

Our regular customers are also provided data which they can use for doing their own design.

Customization

To a large degree we can customize our turbulators to meet your requirement.

Research Assistance

For customers wishing to experiment with new applications using turbulators, we are very happy to share our experience and provide support by way of small customized orders shipped by courier.



Example 1

Example 1 : Brine cooled Air cooler

Here we have as case 1, wire wound fin tube (a special high performance fin tube we make) with no turbulators. In case 2 we have the same wire wound fin tube with internal flexible turbulators. This allows us to see the standalone effect of the turbulators.

Number of rows drops from 20 to 12. Total tube length falls from 1164 to 636. This is a 45% decrease.

Quite naturally the airside pressure drop falls with its commensurate power savings from 1.2 inches of water column to .8 inches. Even more remarkably the tubeside pressure drop falls from .5 to .4 kgs per square cm. This demonstrates that though the pressure drop goes up nominally on account of turbulators the actual total pressure drop can actually fall as the number of passes reduce and the length of tube becomes less.

Example 2 : Produced Water heater.

Here we have case 1 as a normal helical fin tube with no Turbulator and 6 passes and case 2 our own wire wound fin tube (a much higher efficiency fin tube we make) and our flexible Turbulator inside the tube. The results are given as under.

The surface area of fins per linear foot of wire wound fin tube is 4.02 feet against 6.16 for the normal helical fin tube.

The number of tube rows is reduced from 6 to 3 and tube passes from 6 to 1. Total fin surface area is reduced from 30,532 to 11,898. A reduction of 62%.

Example 3 : Shell and Tube exchanger (with Hot Oil on the tubeside and Produced water on the shell side.)

Case 1 is without turbulators and case 2 is with flexible internal turbulators.

The total length of tubes is reduced from 55 lengths of 16 ft to 60 lengths of 10 ft. This reduces the total length of tubes by 32%.

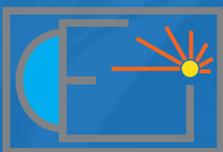
| Item Tag | Brine cooled air cooler Plain Tube(no turbulator) | Brine cooled air cooler (internal flexi turbulator) | |
|---------------------------|---|---|------------------|
| Air flow rate | 7700 | 7700 | m3/hr |
| Air inlet pressure | 2 | 2 | kPa |
| Air inlet temperature | 25 | 25 | deg C |
| Air outlet temperature | 6 | 6 | deg C |
| Heat load | 101910 | 101910 | kcal/hr |
| Brine flow rate | 25 | 25 | m3/hr |
| Brine inlet temperature | -3 | -3 | deg C |
| Brine outlet temperature | 2 | 2 | deg C |
| Fouling factor air side | 0.002 | 0.002 | hr ft2 deg F/btu |
| Fouling factor brine side | 0.001 | 0.001 | hr ft2 deg F/btu |
| Face area length | 1128 | 1128 | mm |
| Face area height | 915 | 915 | mm |
| Tube position | horizontal | horizontal | |
| Tube OD | 19.05 | 19.05 | mm |
| Tube thickness | 16 | 16 | BWG |
| Outside fins | Wire wound | Wire wound | |
| Tube/ Fin material | SS 304 | SS 304 | |
| Fin OD | 45 | 45 | mm |
| Outside heat tr area | 2.45 | 2.45 | ft2/ft |
| Inside turbulator | Nil | 19D3 | |
| Tubes/row | alt 18/17 | alt 18/17 | |
| Rows/pass | 2 | 6 | |
| Inside reynolds no. | 2683 | 896 | |
| jh factor | 7 | 40 | |
| f factor | 0.0004 | 2.11 | |
| LMTDc | 14.2 | 14.2 | deg C |
| hod linear | 42.6 | 42.6 | btu/hr ft deg F |
| hiod linear | 20 | 77.1 | btu/hr ft deg F |
| Ud linear | 13.62 | 27.4 | btu/hr ft deg F |
| Tube length required | 1164.3 | 636.7 | ft |
| Tube length/row | 63.2 | 63.2 | ft |
| No. of rows reqd | 18.4 | 10.1 | |
| No. of rows provided | 20 | 12 | |
| No. of passes | 10 | 2 | |
| Heat tr area provided ext | 3173.4 | 1856.3 | ft2 |
| Air side pressure drop | 1.2 | 0.8 | in WC |
| Brine side pressure drop | 0.5 | 0.4 | kg/cm2 |

Example 2

| Item | Produced water cooler | Produced water cooler | |
|---------------------------|-----------------------|-----------------------|------------------|
| Tube side flow rate | 47354 | 47354 | lbs/hr |
| Tube side inlet temp. | 246 | 246 | deg F |
| Air flow rate | 293602 | 293602 | m3/hr |
| Air inlet temperature | 95 | 95 | deg F |
| Air inlet pressure | 100 | 100 | mm Wc |
| Air outlet temperature | 127.9 | 127.9 | deg F |
| Tube side outlet temp. | 120 | 120 | deg F |
| Heat load | 6248651 | 6248651 | btu/hr |
| Fouling factor air side | 0.002 | 0.002 | hr ft2 deg F/btu |
| Fouling factor tube side | 0.002 | 0.002 | hr ft2 deg F/btu |
| Face area length | 20 | 20 | ft |
| Face area width | 9.524 | 9.524 | ft |
| Tube position | horizontal | horizontal | |
| Tube length | 20 | 20 | ft |
| Tube OD | 1 | 1 | in |
| Tube thickness | 14 | 14 | BWG |
| Outside fins | Plain round | Wire wound | |
| Tube material | Carbon steel | Carbon steel | |
| Fin material | Aluminium | GI wire | |
| Fin OD | 57.1 | 51.1 | mm |
| Outside heat tr area | 6.16 | 4.02 | ft2/ft |
| Pitch in rows | 67 | 57 | mm |
| Row pitch | 58 | 44 | mm |
| Tubes/row | alt 43/42 | 50 | |
| Rows/pass | 1 | 3 | |
| Inside turbulator | Nil | 25D3 | |
| Ud linear | 28.24 | 46.06 | btu/hr ft deg F |
| Ud | 4.584 | 11.46 | btu/hr ft2 deg F |
| | 22.37 | 55.9 | kcal/hr m2 deg C |
| Tube length required | 4452.1 | 2619 | ft |
| Tube length/row | 838.6 | 986.6 | ft |
| No. of rows reqd | 5.31 | 2.65 | |
| No. of rows provided | 6 | 3 | |
| No. of passes | 6 | 1 | |
| Heat tr area provided ext | 30532.3 | 11898.7 | ft2 |
| | 2837.6 | 1105.8 | m2 |

Example 3

| Item | Hot oil | Hot oil | |
|---------------------------|----------------|----------------|------------------|
| Shell side fluid | Produced water | Produced water | |
| Tube side flow rate | 19634 | 19634 | lbs/hr |
| Tube side inlet temp | 550 | 550 | degF |
| Shell side flow rate | 47536 | 47536 | m3/hr |
| Shell side inlet temp | 175 | 175 | deg F |
| Shell side outlet temp | 246 | 246 | deg F |
| Tube side outlet temp | 266 | 266 | deg F |
| Heat load | 3573838 | 3573838 | btu/hr |
| Fouling factor shell side | 0.003 | 0.003 | hr ft2 deg F/btu |
| Fouling factor tube side | 0.002 | 0.002 | hr ft2 deg F/btu |
| Allowable pr drop shell | 5 | 5 | psi |
| Allowable pr drop tube | 10 | 10 | psi |
| Heat exchanger type | BEU | BEU | |
| Shell ID | 19 | 19 | in |
| Tube type | plain | plain | |
| Tube OD | 1 | 1 | in |
| Tube thickness | 14 | 14 | BWG |
| Tube material | S32205 Duplex | S32205 Duplex | |
| Tube pitch | 1.25 | 1.25 | in |
| No of passes | 8 | 6 | |
| No of tubes [U bends] | 55 | 60 | |
| Baffle type | segmental | segmental | |
| Baffle spacing | 6 | 6 | in |
| Inside turbulator | nil | 25D2 | |
| Outside coeff ho | 768.3 | 705.9 | btu/hr ft2 deg F |
| Inside coeff hi | 125.8 | 290.2 | btu/hr ft2 deg F |
| Overall coeff clean Uc | 86.52 | 159.5 | btu/hr ft2 deg F |
| Overall service Ud | 58.9 | 85.7 | btu/hr ft2 deg F |
| | 287.4 | 418.2 | kcal/hr m2 deg C |
| LMTDc | 156.5 | 156.9 | deg F |
| Tube length provided | 16 | 10.0 | ft |
| Heat tr area prov. ext | 470.6 | 325.2 | ft2 |
| | 43.7 | 30.2 | m2 |
| Overdesign | 21.45 | 22.35 | % |
| Shell side pr drop | 1.735 | 1.224 | psi |
| Tube side pr drop | 2.428 | 9.483 | psi |



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